

# B-SERIES BALL SCREW ACTUATOR



LINEAR MOTION  
800-323-9114

## B-SERIES MOTOR SIZING AND TORQUE CALCULATIONS (Table 8)

MODEL No.	GEAR RATIO	Turns per inch TPI	Rated cap (lbs) P	STATIC TORQUE T <sub>s</sub>	UNIT INPUT TORQUE AT RATED CAPACITY								NO LOAD TORQUE T <sub>o</sub>
					T = RUNNING TORQUE (lb-in) at VARIOUS RPM (Theoretical)								
					50 RPM	115 RPM	172 RPM	345 RPM	600 RPM	870 RPM	1140 RPM	1725 RPM	
B-1	5:1	10	2,000	49	44	43	42	41	40	40	39	38	3
	10:1	20		29	26	25	24	23	22	22	21	21	
B 2.5	6:1	24	5,000	41	37	36	35	34	33	33	32	32	5
	24:1	96		17	14	13	13	12	11	11	10	10	
B 5	6:1	12.66	10,000	186	170	167	164	160	156	154	152	150	12
	24:1	50.66		71	59	57	54	51	48	47	45	43	
B 10	8:1	16.88	20,000	302	269	262	256	247	240	236	233	229	18
	24:1	50.66		154	125	118	112	105	98	94	91	87	
B 20	8:1	16	40,000	628	561	546	533	516	502	493	487	479	36
	24:1	48		313	254	242	230	215	202	194	189	181	
B 30	10 2/3:1	16	60,000	987	864	838	815	784	761	746	736		48
	32:1	48		520	408	384	363	335	312	298	289		
B 50	10 2/3:1	16	100,000	2519	2149	2105	2010	1929	1870	1835	1814		96
	32:1	48		1361	1017	948	887	811	755	722	631		
B 75	10 2/3:1	10.66	150,000	3604	3132	3036	2952	2846	2769	2723	2335		156
	32:1	32		1781	1384	1303	1232	1141	1074	1035	1010		
B 100	12:1	12	200,000	4622	3844	3696	3567	3413	3307	3248	3212		204
	36:1	48		2568	1837	1697	1575	1429	1327	1271	1236		

For RPM's not shown use the next slowest RPM. For speeds less than 50 RPM contact factory.

10. Determine Uni-Lift Running Load Proportion Factor: (f)

$$f = \frac{P_3}{(P \times N)}$$

P = Rated Capacity of Uni-lift

P<sub>3</sub> = Max. system running load N= Number of Uni-lifts

11. Determine Unit Running Torque: (T<sub>1</sub>) (lb-in)

$$T_1 = (T \times f) + T_o$$

T<sub>o</sub> = No load torque from chart

T = Running torque from chart

12. Find the System Running Torque: (T<sub>2</sub>) (lb-in)

$$T_2 = \frac{(T_1 \times N)}{e_1}$$

e<sub>1</sub> = System Arrangement Efficiency, see page 77

13. Find System Power:

$$\text{System HP} = \frac{(T_2 \times \text{RPM})}{(63025 \times e_2)}$$

e<sub>2</sub> = Reducer Efficiency, see page 77

RPM = Uni-Lift input shaft speed

14. Determine System Starting Torque: (T<sub>s2</sub>)

$$T_{s2} = \frac{((T_s \times f) + T_o) \times N}{e_2}$$

T<sub>s</sub> = Static torque from chart

15. Determine Motor Starting Torque: (T<sub>sm</sub>) (lb in)

$$T_{sm} = \frac{T_{s2}}{(R \times e_2)}$$

R = Gear Reducer Ratio

16. Determine Motor Running Torque: (T<sub>rm</sub>)

$$T_{rm} = \frac{T_2}{(R \times e_1)}$$

- Select a motor with a power rating greater than HP requirement in step 13, a starting torque greater than T<sub>sm</sub> requirement in step 15, and a motor running torque greater than T<sub>rm</sub> in step 16.

- Select system torque transmission equipment (reducer, mitre gear boxes, couplings, etc.) with ratings greater than the torque to be transmitted, see step 12 and system arrangements, page 77.

- Size shafting for system starting torque to be transmitted, see step 16, and Table B page 76.

17. Select a Brake Size (required for all ball screw jack applications):

$$T_b = \frac{C}{\text{TPI} \times d \times R} \times \frac{(f \times T_{hb} \times N)}{R}$$

C = Motor brake factor

R<sub>1</sub> = Reducer ratio

T<sub>b</sub> = Motor brake torque

T<sub>hb</sub> = Hold torque (see technical data)

d = Stopping distance

N = Number of Uni-Lifts in the system

### C Factor For Motor Brake (ft.-lbs.)

Motor	1140	1725	Motor	1140	1725
1/4	3.2	4.1	3	38	66.5
1/3	4	4.9	5	48.3	87.4
1/2	5.1	6.1	7 1/2	69.4	112
3/4	7.89	9.2	10	126	146
1	9.18	17.8	15	268	273
1 1/2	11.3	21.6	20	306	315
2	29.5	25.6	25	548	596